

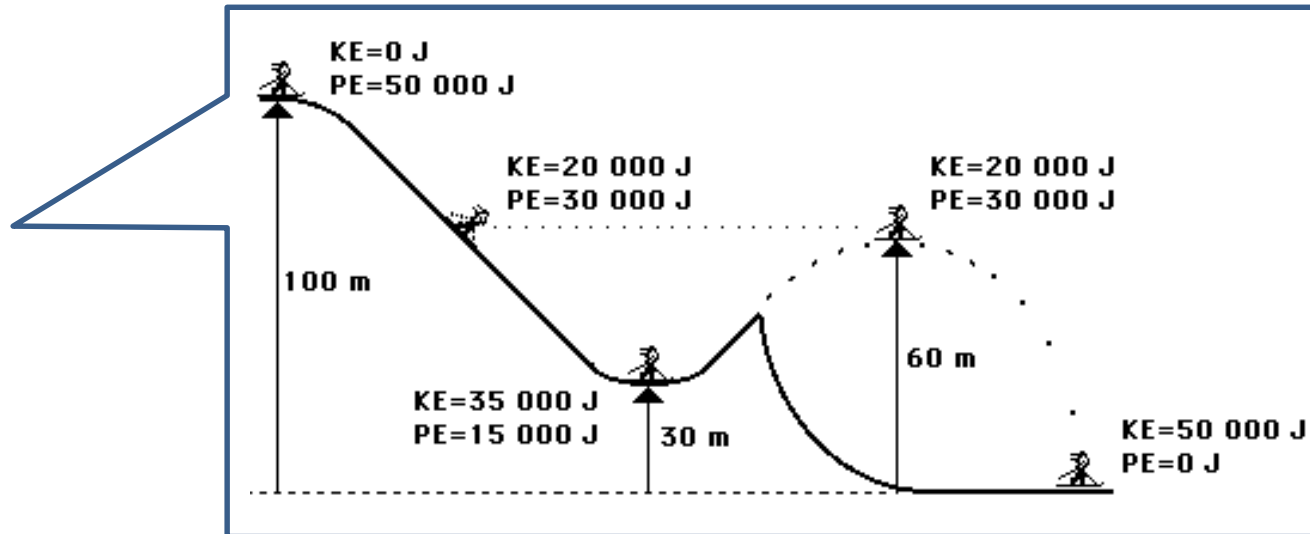
Advanced Energy System Eng.

Definition & forms of energy

- **Energy: the capacity or capability to do work**

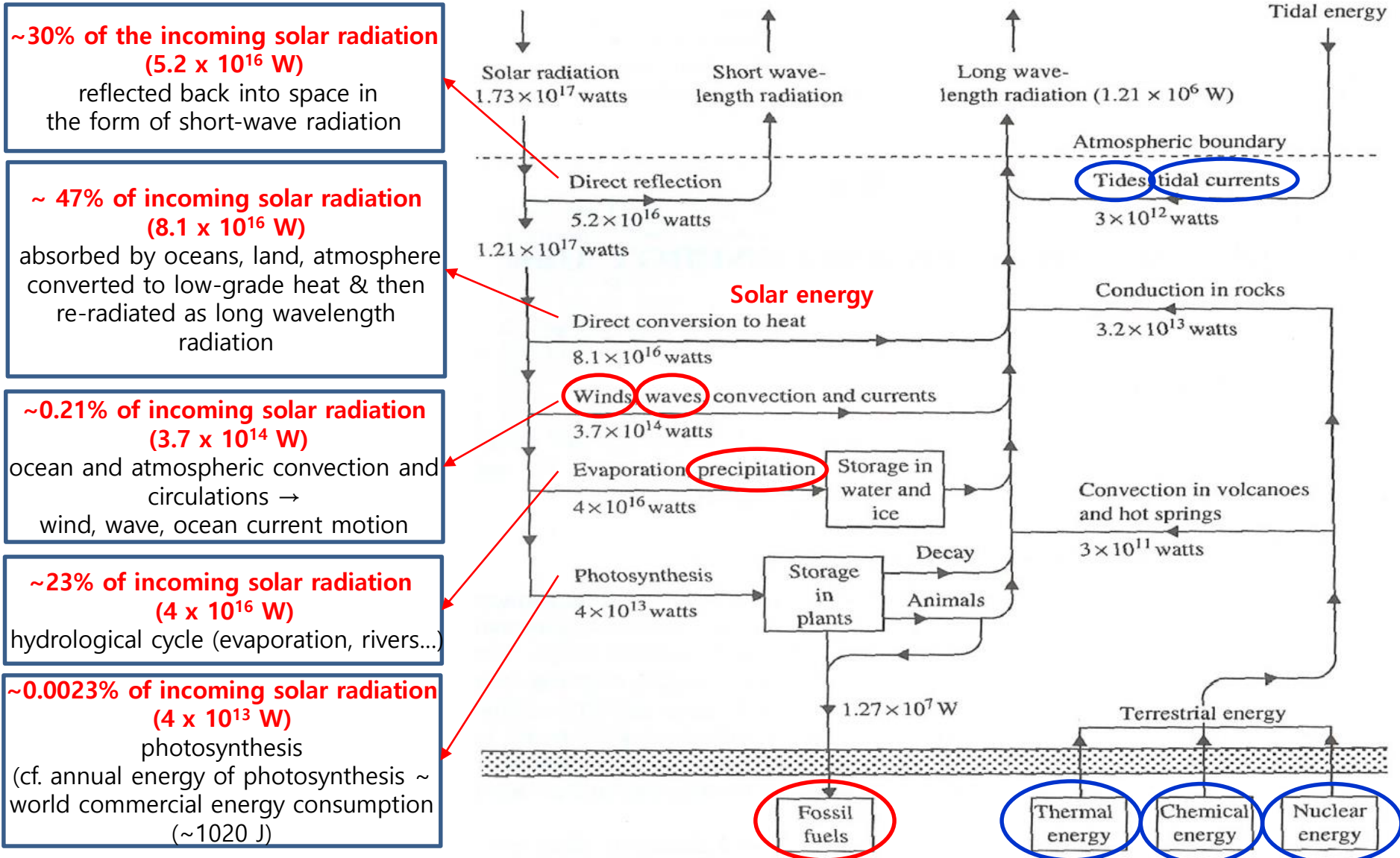
- **Forms of energy**

- biofuels (e.g., wood)
- chemical
- electrical
- mechanical-kinetic
- mechanical-potential
- gravitational
- nuclear
- heat (thermal)
- radiation
- magnetic
- sound



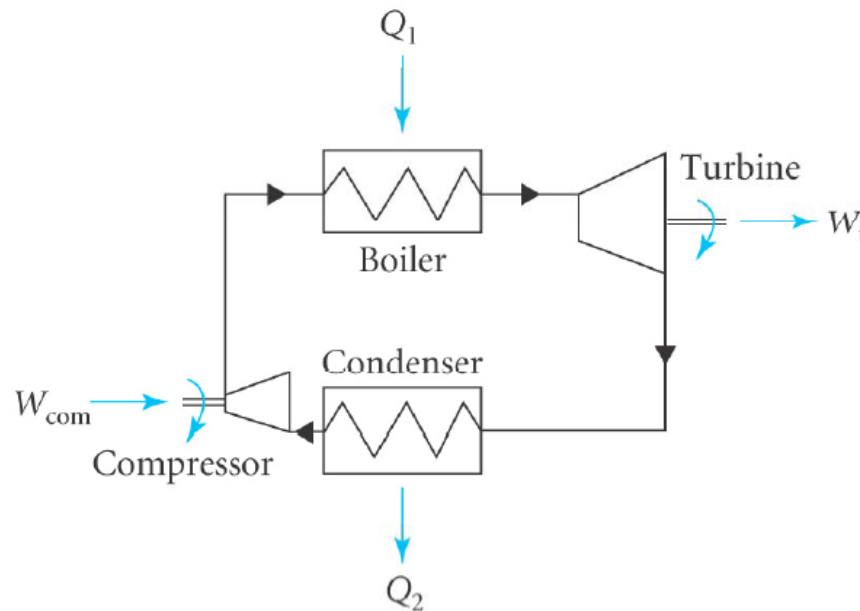
- **Energy conversion:** transformations between different forms of energy
 - conversion efficiency $\rightarrow$ input > output : < 100%
- **e.g.,** solar cell ~ 10% (radiant to electrical)
 - automobile engine ~25% (chemical to thermal, thermal to mechanical)
 - fuel cell ~60% (chemical to electrical)
 - secondary battery ~75% (chemical to electrical, electrical to chemical)

Energy flow upon the Earth from natural sources



4. Closed cycle for a steam power plant

- (1) Compressor (boiler feed pump): work W_{com} done on the system to compress cold **water** from subatmospheric pressure to high pressure
- (2) Boiler: heat Q_1 added to the system to convert cold water into steam
- (3) Turbine: work W_t done by the system (i.e. by steam) on the turbine blades
- (4) Condenser: heat Q_2 lost from the system to the environment in converting steam back to cold water



Thermal Energy

After each complete cycle, the working fluid has the same internal energy, U

→ net change in internal energy is zero, $\Delta U = 0$

$$(Q_1 - Q_2) - (W_t - W_{com}) = 0$$

The **efficiency** of the process, $\eta = [\text{net work output}]/[\text{heat input}]$

$$= (W_t - W_{com})/Q_1$$

$$= (Q_1 - Q_2)/Q_1$$

$$= 1 - (Q_2/Q_1)$$

$Q_1 \text{ \& } Q_2 > 0 \rightarrow \eta < 1$: system is less than 100% efficient

→ “**2nd law of thermodynamics**”: heat increases the disorder (entropy), **no** system operating in a closed cycle can convert all the heat absorbed from a heat reservoir into the same amount of work

Carnot: maximum possible efficiency of a heat engine operating in a closed cycle between two heat reservoirs depends only on the ratio of the absolute temperatures of the reservoirs (*s-T-s-T constant*)

$$\eta_c = 1 - (T_2/T_1)$$

T_1 and T_2 are the absolute temperatures (K) of the upper & lower reservoirs

Thermal Energy

Efficiency of a Carnot cycle

Consider ideal gas ($pV = nRT$) operating in the closed cycle $abcda$

ab & cd are isotherms at T_1 & T_2 ;

bc & da are reversible adiabatics (i.e., no heat transfer with the surroundings)

By the 1st law of thermodynamics,

$$dQ = dU + pdV = C_v dT + pdV$$

(C_v : specific heat at constant volume)

Along isotherm ab , $dT = 0 \rightarrow dQ = pdV$

$$Q_1 = \int pdV = nRT_1 \int (dV/V) = nRT_1 \ln(V_b/V_a)$$

Along cd , $Q_2 = nRT_2 \ln(V_c/V_d)$

$$Q_2/Q_1 = [T_2 \ln(V_c/V_d)] / [T_1 \ln(V_b/V_a)]$$

Along adiabatic da , $dQ = 0 \rightarrow C_v dT = -pdV$

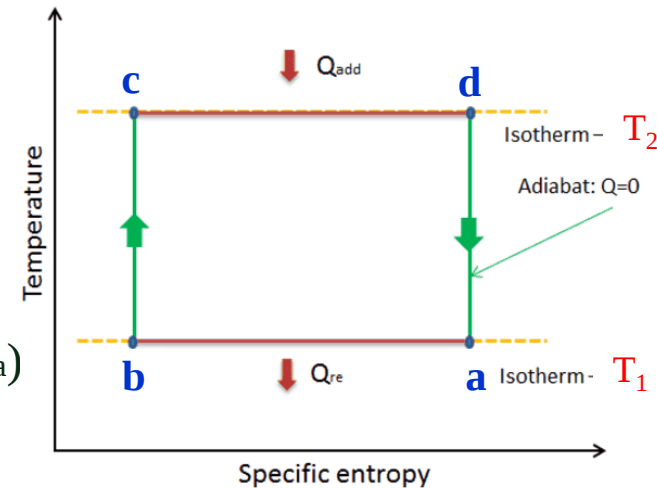
$$\rightarrow C_v (dT/T) = -nR (dV/V)$$

$$\rightarrow C_v \ln(T_1/T_2) = -nR \ln(V_a/V_d)$$

Along adiabatic bc , $C_v \ln(T_1/T_2) = -nR \ln(V_b/V_c)$

$$\rightarrow (V_a/V_d) = (V_b/V_c) \text{ or } (V_c/V_d) = (V_b/V_a)$$

$$\rightarrow Q_2/Q_1 = T_2/T_1 \quad \eta = 1 - (Q_2/Q_1) = 1 - (T_2/T_1)$$

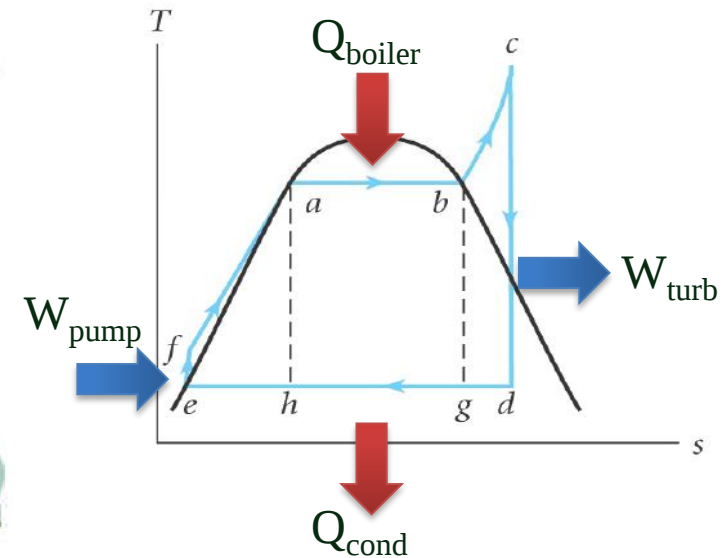
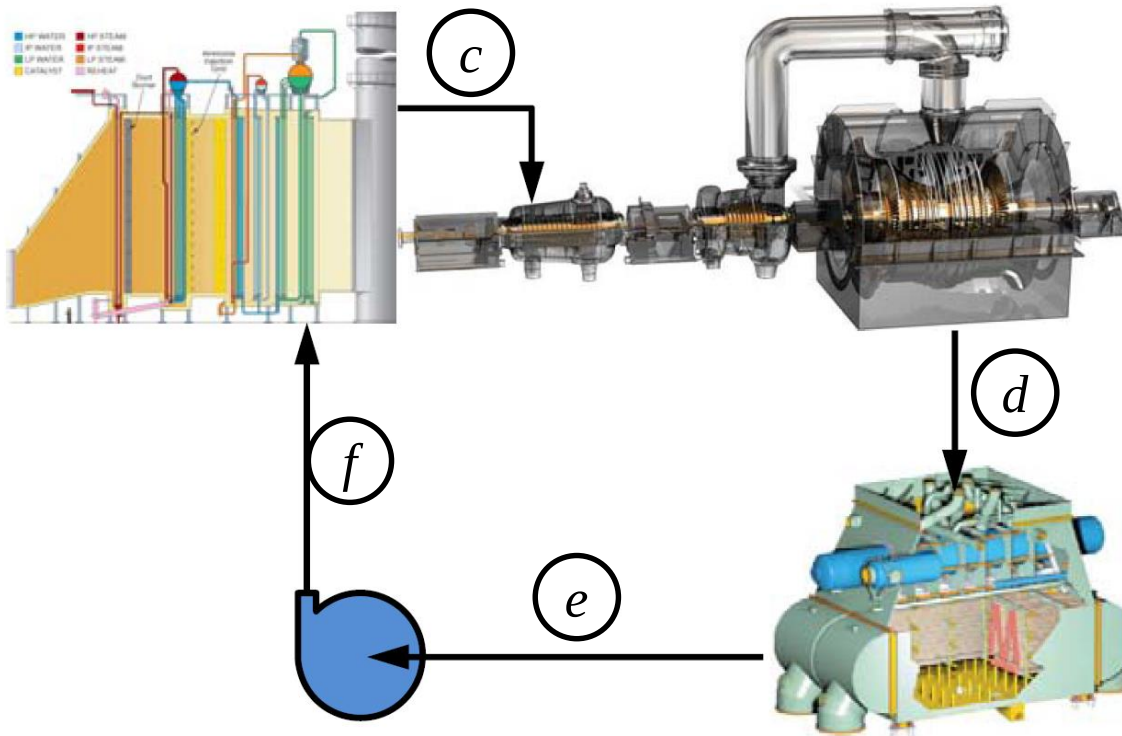


6. Disadvantages of a Carnot cycle for a steam power plant

- T_1 of the upper reservoir to be constant
 - only achieved along an isobar in region II
 - not possible to operate the boiler in the dry steam region (III) since T rises
 - upper temperature of the cycle, T_1 , is constrained by the maximum temperature of the two-phase boundary
- Another problem arises in the turbine: turbine operates with a two-phase mixture of water and steam (region II) → momentum of fast moving water droplets in the mixture damages the turbine blades and shortens the life of the turbines
- Same in compressor: damage the blades of the compressor.
Volume of steam in the mixture is very large → large compressor needed so expensive
- Carnot cycle: impractical for a steam power plant

Thermal Energy

7. Rankine cycle for steam power plants



Thermal Energy

(2) Rankine cycle with reheat

Reheat is becoming common for the large combined cycle power plants, where inlet pressures and temperatures are rising to those commonly found in traditional fossil power plants.

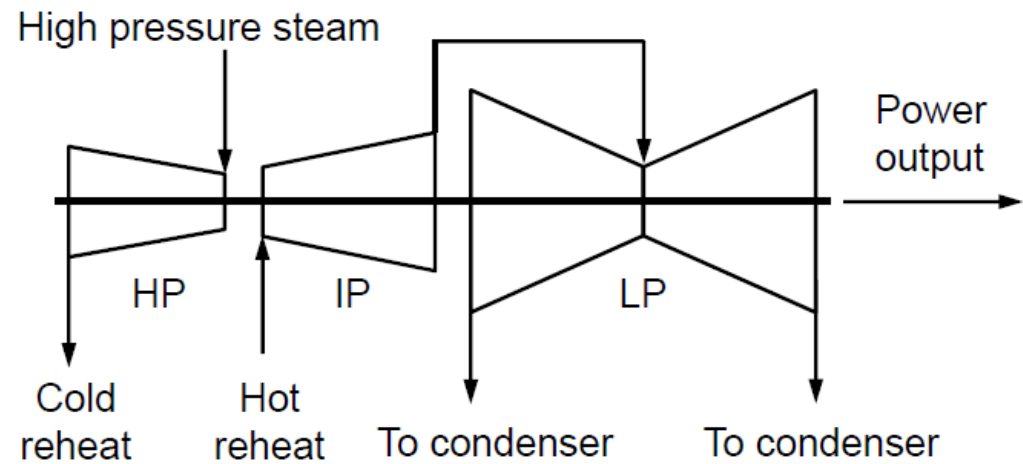
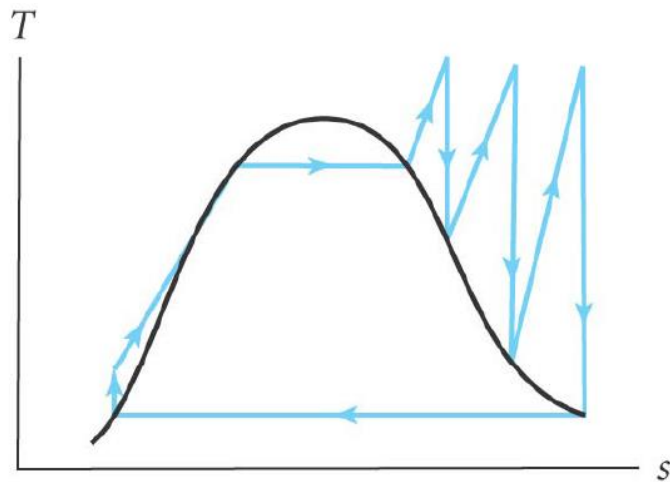
Rankine cycle without reheat: *difficult to completely eliminate water droplets* which have high momentum and wear out the turbine blades

→ "with reheat" is introduced → steam is reheated several times before entering the condenser

→ increase efficiency and eliminate water droplets

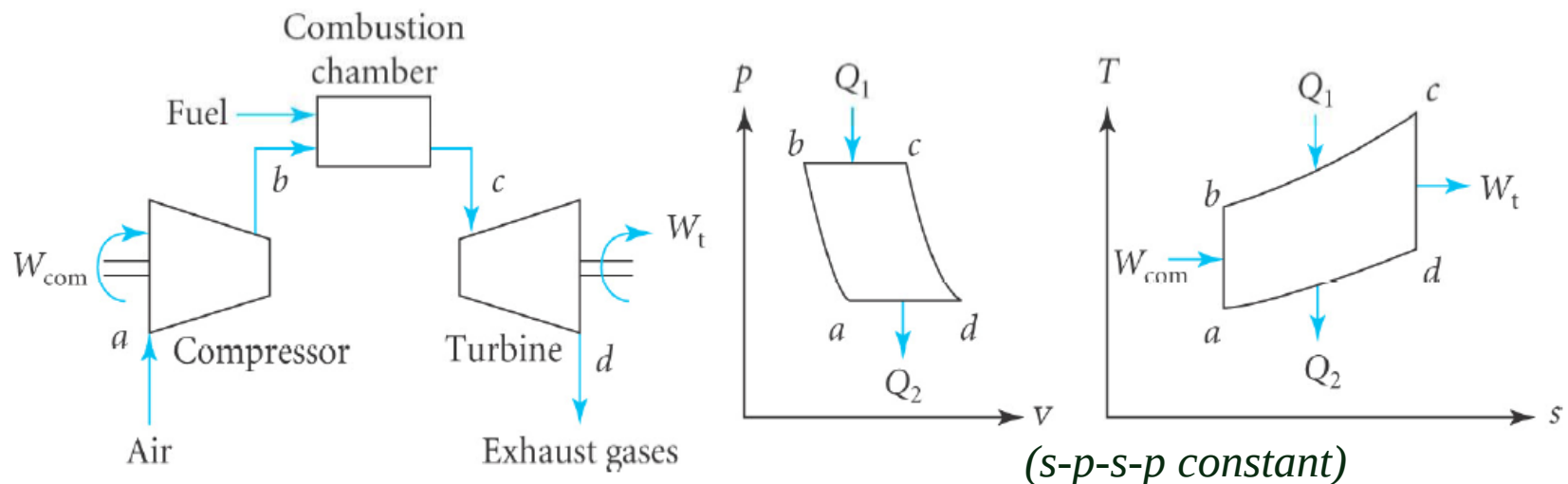
Maximum operating temperature in the superheater: 650°C (손상방지를 위해)

Efficiency of the state-of-the-art technology: 40~45%



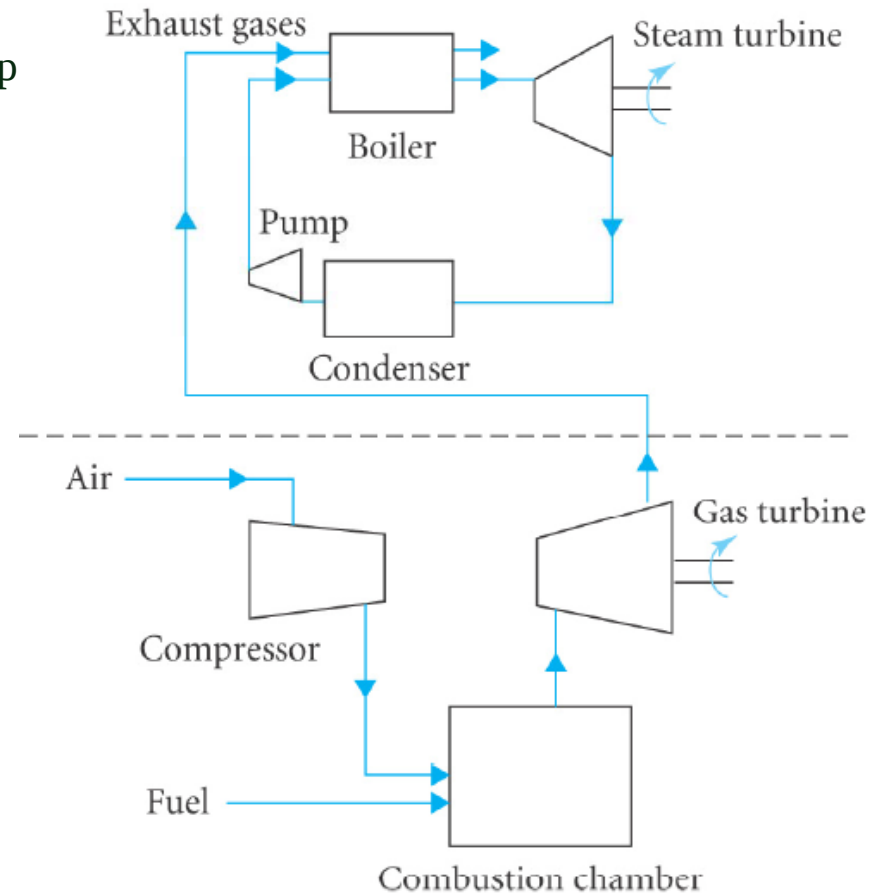
8. Gas turbines & the Brayton (or Joule) cycle

- Gas turbine: gaseous products of combustion is $1100 \sim 1600^{\circ}\text{C}$ (ceramic coating on turbine blades, thermal barrier coating, TBC)
- Gas turbine for power generation is originated from the jet turbine engine
- Jet engine: combustion of gaseous fuel & expansion of exhaust gases
 - same phase, no condenser,
 - smaller and cheaper than steam engine
- Brayton (or Joule) cycle: atmosphere acts as the heat exchanger → **closed cycle**



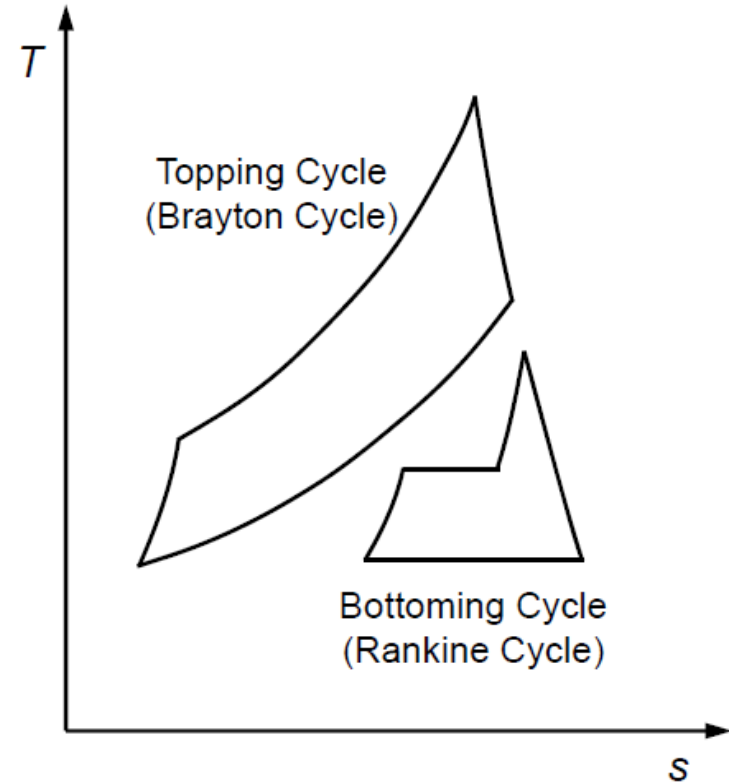
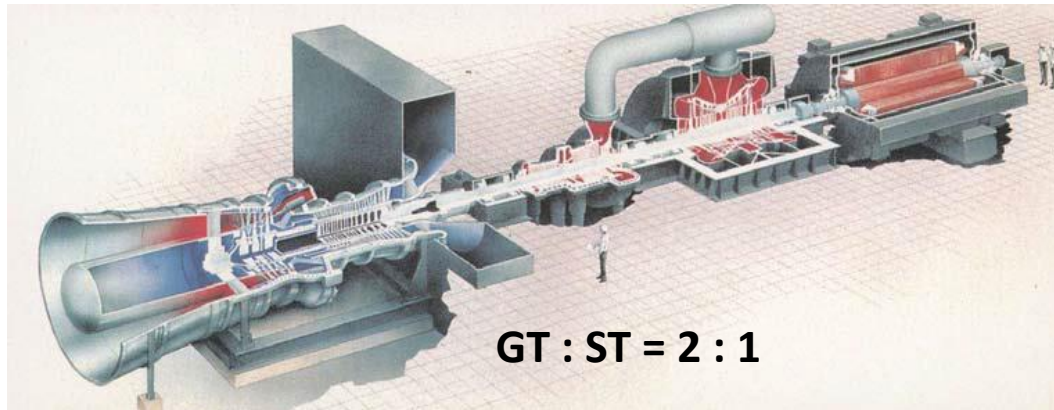
9. Combined cycle gas turbine (CCGT)

- Hot exhaust gases from gas turbine used to heat up the water for steam power plant (HRSG, heat recovery steam generator)
- Increase the cycle efficiency: CCGT
→ $\eta \sim 60\%$
- Combined heat and power cycle (CHP)
→ $\eta \sim 80\%$
 - condenser is generally operated **at the higher temperature** than steam power plant
 - Waste heat is used for the local heating nearby
 - Disadvantage: expensive → can be used for highly populated area or industrial complex

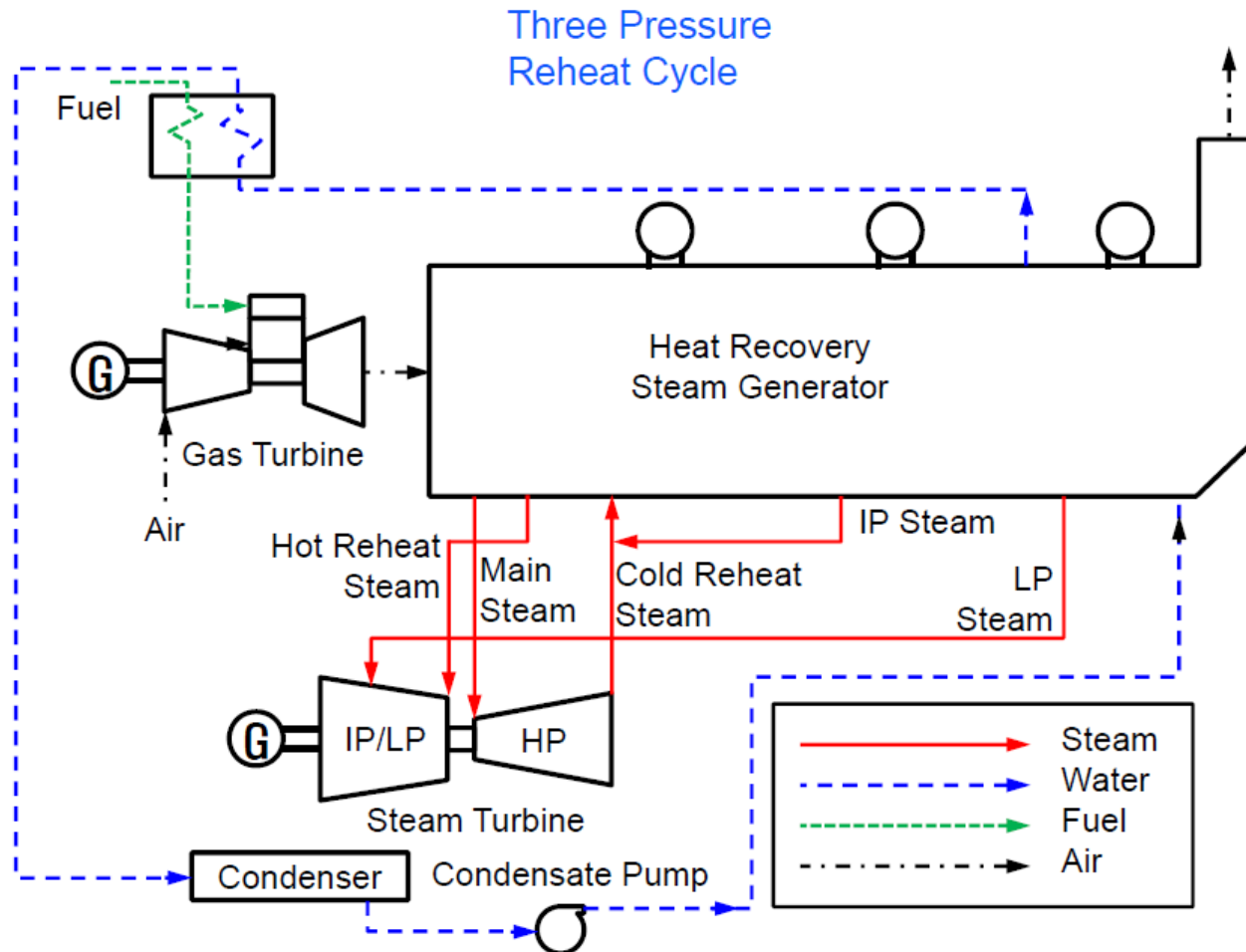


Thermal Energy

9. Combined cycle gas turbine (CCGT)

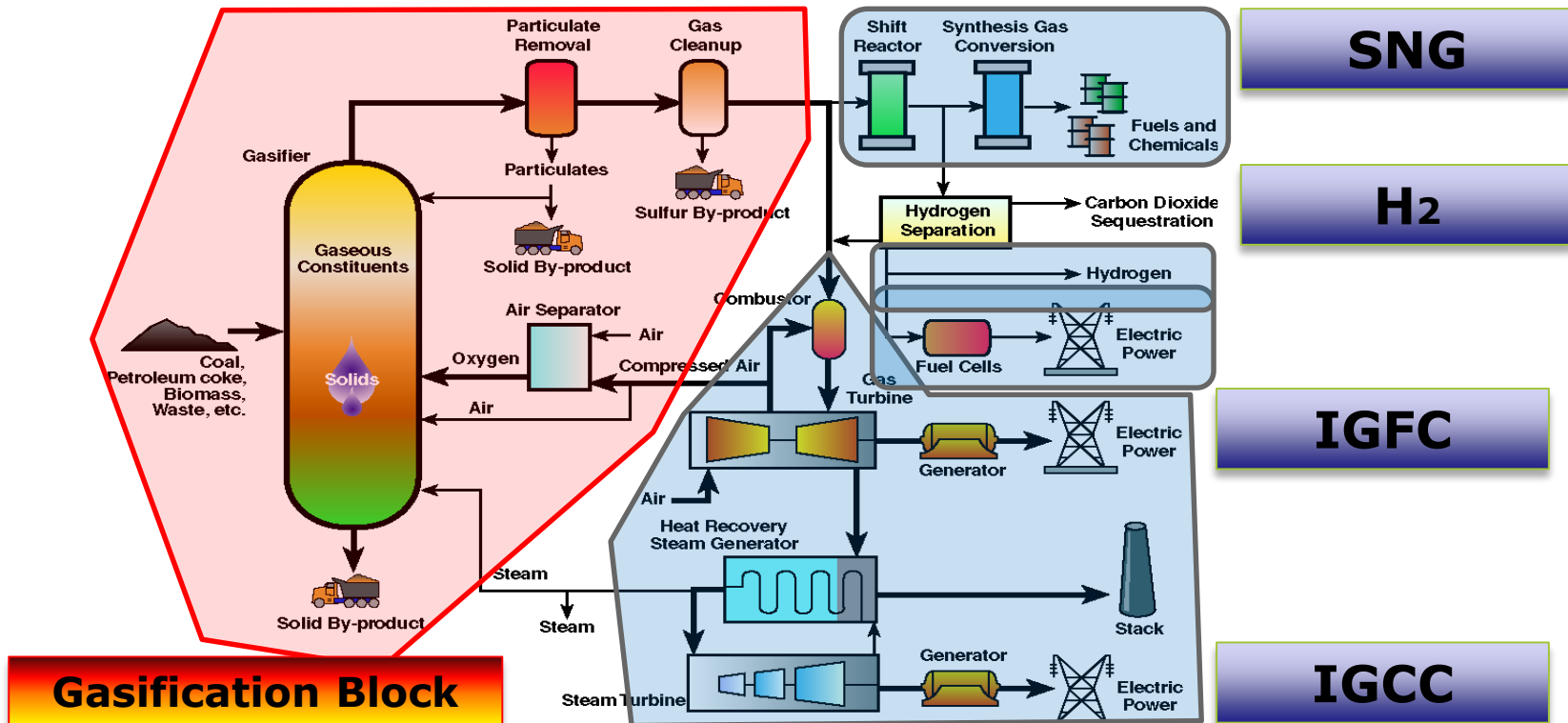


9. Combined cycle gas turbine (CCGT)



12. Gasification

- IGCC (Integrated Gasification Combined Cycle)
- IGFC (Integrated Gasification Fuel Cell)
- SNG (Synthetic Natural Gas or Substitute Natural Gas)



Thermal Energy

- IGCC can be efficiently combined with carbon capture and storage (CCS)

